

#### M. P. LEVEL on behalf of the SOLEIL team



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- Parameters of the Storage ring
- ≻Orbit correction scheme
- ≻Stability criteria
- ≻Foundations:
  - APD solution
  - Vibration measurement campaigns
  - New foundations solution
- Strategy for the SR:
  - Medium and short term stability
  - Slow feedback, fast feedback and feed-forward compensation



Energy:  $2.75 \; GeV$ Circumference: 354.097m*Emittance (rms):* 3.70 nmrad Number of cells / super periods: 16/412m x 4 ; 7 m x 12 ; 3.8 m x 8 Straight sections: Betatron tunes,  $Q_{\chi}/Q_{\nu}$ : 18.2 / 10.3 Natural Chromat.  $\xi_x/\xi_v$ : - 2.88/-2.21 Momentum compaction:  $4.49 \times 10^{-4}$ 1.02 10-3 Energy dispersion





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120 located close to the Quad. or Sext. and Straight S.

- Closed Orbit Distortion:
- 56 girders of 3 types
- Standard alignment errors  $\Rightarrow$ COD max : 15 mm (H), 7 mm (V)
- Closed Orbit Correction: 120 possible correctors in Sext.
- SVD method: 56 eigenvalues in H and V planes
- Minimal configuration: 56 correctors in H and V planes
- H plane: Max. value=500 $\mu$ m, always in the BMs (will be improved after magnetic meas.) Straight sections: r.m.s. = 25  $\mu$ m and maximum 100  $\mu$ m
- V plane: Max. value=150µm, always in the BMs Straight sections: r.m.s. =15 µm and maximum 100 µm

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taking into account BPM displacement errors of 100μm - r.m.s.
 and output accuracy of 0.2 μm - r.m.s.

	Horizontal plane	Vertical plane
Maxium rms value	0.133mrad	0.0937mrad
Maximum value	0.401mrad	0.238mrad

Remark: BPM position error will be reduced to 50 µm r.m.s. by B.B. A.
✓ It was decided to take a factor 2 margin and to design the maximum corrector deflexion at 0.8 mrad.
✓ The resolution is not completely decided between 18 and 20 bits (We must also take into account the noise induced by SVD algorithm)



## **Different noise sources with 3 different time scales:**

- Long term stability: differential settlement and temperature variation with seasons  $\Rightarrow$  Building foundations
- Medium term stability: in general thermal drifts  $\Rightarrow$  vacuum chamber, water cooling, air conditioning etc
- •Short term stability :
  - random in-situ sources of vibrations surrounding the machine site, such as:human activities, mechanical devices, water cooling,...
    external source of vibration generating plane waves propagating in the ground.



The detrimental effect of beam position instabilities can be seen as a macroscopic increased emittance. Following the users specification we take:

 $\sigma_{COD}$  < 0.1  $\sigma_{Beam}$  and  $\sigma'_{COD}$  < 0.1  $\sigma'_{Beam}$ 

	σ <sub>cob</sub> (μm)	σ' <sub>COD</sub> (µrad)	
Horizontal	18	3	
Vertical	0.8	0.5	

In the most stringent case this correspond to  $\Delta \varepsilon / \varepsilon = 1\%$ 

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## **Building foundations: Design criteria**



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# **Building foundations: APD98 solution**



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## •Solution APD 98, reservations about:

-Elastomer supports: *behaviour with time?* No *possibility to inspect it.* 

-The differential settlement between the ring tunnel and experimental hall slab: *it should be gone beyond the deformation criteria (>100 microns/year during 10 years) due to the swelling effect of silt and clay* 

-The justification of the advantage of the piles with sleeving



•3 campaigns of vibrations measurements coming from the site environment showed:

- –Sinusoidal noise with in the daytime a maximum elongation peak to peak of 0.35  $\mu$ m + some accidents of 0.5 0.7  $\mu$ m (planar wave 2.5 Hz)
- -same amplitude level at the ground level and -15m deep at the Fontainebleau sands level.
- -The accidents have been identified as waves produced by some kind of public work trucks in correlation with some kind of irregularities of the two adjacent roads





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**Exemple of localization of one** event of amplitude > 0.45 microns peak to peak





AVLS/VI571 Fichier TC105.tim canal vertical

We notice that the lorry produces essentially a frequency of 2.5 Hz (0.57 micron peak to peak)



### The confirmation of these results have been realized by organising night traffics of 4 different types of heavy trucks on the roads lining the site:

 $\Rightarrow$ The importance of the vibratory levels reached is not only linked to the trucks weight but essentially depends on the closeness of the two frequencies: the truck suspension resonance frequency and the typical frequency of the ground, of the order of 2.5 Hz.

 $\Rightarrow$ Another factor plays a part: the longitudinal profile which properties favour the vibratory exitations in the particular frequency range : The average truck speed being of 60 km/h, the most unfavorable road defect is the one that generates an excitation of 2.5 Hz, which corresponds to a 3 meter long basin

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 $\Rightarrow$  Led to plan a repairs of the roadway. The maintenance of the quality of their surface in the time will be necessary to get us rid of these vibrations.

 $\Rightarrow$  Set again the question about the interest of the double sleeving of the piles bearing the slab of the storage ring tunnel.

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Design of beam line loading

### global load: 60T on average



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 $\Rightarrow$  Solution of APD98 cannot reach criteria on this site.

- $\Rightarrow$  2 families of solutions studied :
- Bored piles under the slabs of the ring tunnel and experimental hall:
  - with double sleeving
  - without sleeving
- Compacted soil (continuous slab put directly on it):
  - Substitution of a part of the silt by a sand-gravel-cement mixture (3m height)

### $\Rightarrow$ Analysis of the static behaviour:

-the intrinsic weight (slab and local storage ring tunnel, slab of the experimental hall) -the load (addition of the load due to a beam line installation)



• The two kinds of solutions enable to respect the criteria with the « noble » area (experimental slab, ring, booster and linac)

In a budget point of view the solution piles without sleeving is by far the best one :
 soil reconstructed solution : +3M€ vs the APD 98 solution

solution piles with sleeving: +4.6M€ solution piles without sleeving : +1.2M€

# Chosen solution: bored piles

•Slab (0.8m thick) of the ring tunnel and experimental hall on simple bored piles (diameter 0.8 and 0.6m respectively) *with connected slab* 

•No treatment of the soil

420 under the experimental hall (4\*105)
128 under the ring tunnel
64 under linac and booster with a slab unconnected



No differential settlement between the ring tunnel and experimental hall.When a new beam line is installed: the criteria are respected:

	Displacement (µm)		
Location	immediately	after 6 months	total
A ring	4	8	10
A' ring	13	27	35
B beam line	15	31	40
C beam line	41	55	110
Maxi under beam line	58	120	155
B' neighbouring b.l.	19	39	50
C' neighbouring b.l			0

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## ⇒Dynamic studies

Software SASSI: impedance of piles (stiffness, damping)Software ANSYS: answer of a structure to a dynamic request (piles simulated by 3 springs)

•The ground shows an amplification at 2.5; 5;9 and 12 Hz

•The structure presents an amplification at 15 Hz

•The simulation of a propagating wave (same characteristics as those measured at the center of the ring) 2.5 Hz; amplitude 0.7 µm peak to peak  $\Rightarrow$  slab displacement: 0.78 µm peak to peak  $\Rightarrow$  no amplification by the slab

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## Medium term stability: in general thermal drifts

- •Space of 2 mm between magnetic elements and vacuum chamber.
- •Air conditioning in the tunnel and water cooling :  $T = 21^{\circ}C$ (±0.1°). Air conditioning in the experimental hall :  $21^{\circ}C$  (±1°).
- •BPM rigidly fixed on the girder with Thermal screen



## Design of the girder

• Static: Alignment specifications

HLS network

- $\Rightarrow$ 3 jacks ; 4 supports all in the upper part of the girder
- Dynamic: Locking system  $\Rightarrow$  high first resonance mode (42 Hz)

 $\Rightarrow$ No amplification of the propagating wave (0.78 µm peak to peak; 2.5 Hz) by the girder





ANSYS 6.1 NOV 20 2002 09:52:54 DISPLACEMENT STEP=1 SUB =1 FREQ=46.82 PowerGraphics EFACET=1 AVRES=Mat DMX =.020499 \*DSCA=31.687 XV =-.55356 YV =-.78062 ZV =.29019 \*DIST=5.409 \*XF =-1.48 \*YF =-3.988 \*ZF =.38929 A-ZS=68.039 Z-BUFFER EDGE



ANSYS 6.1 NOV 20 2002 09:54:25 DISPLACEMENT STEP=1SUB = 2FREQ=47 PowerGraphics EFACET=1 AVRES=Mat DMX =.021108 \*DSCA=30.773 XV =-.55356 YV =-.78062 ZV =.29019 \*DIST=5.409 \*XF =-1.48 \*YF =-3.988 \*ZF =.38929 A-ZS=68.039 Z-BUFFER



ANSYS 6.1 NOV 20 2002 09:55:13 DISPLACEMENT STEP=1SUB =3 FREQ=54.067 PowerGraphics EFACET=1 AVRES=Mat DMX =.01559 \*DSCA=41.666 XV =-.55356 YV =-.78062 ZV =.29019 \*DIST=5.409 \*XF =-1.48 \*YF =-3.988 \*ZF =.38929 A-ZS=68.039 Z-BUFFER EDGE



 $\sigma_{COD} < 0.1 \sigma_{Beam}$  and  $\sigma'_{COD} < 0.1 \sigma'_{Beam} \Rightarrow$  in the most stringent case tolerance for emittance growth:  $\Delta \epsilon / \epsilon + 1\%$ 

For an emittance coupling of 1% :

**Control ted motion :** the r.m.s. values of the tolerances on the girders motion are 3.9 mm peak to peak and 1mm peak to peak respectively in the horizontal and the vertical plane.

**Correlated motion :** The tolerated vertical amplitude corresponding to 2.5Hz (*vibration measurements performed on the SOLEIL site*) is of 1.25  $\mu$ m peak-to-peak which is above the amplitude of the cultural noise measured on the site (0.35  $\mu$ m peak-to-peak, with some accidents at 0.7 $\mu$ m peak-to-peak).

Remark: for a coupling of 0.1% the tolerated vertical amplitude is 0.4  $\mu$ m peak-to-peak, so it is necessary to suppress the accidents of higher amplitude

#### Vertical peak wave amplitude tolerated



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## Feedbacks

➢ Global slow feedback using 120 BPMs and 56 correctors in each planes H and V:

•resolution 0.2  $\mu$ m, correction every second (cf. J.C. Denard)

• bandwidth 0 to 0.1 Hz

> Global fast feedback  $\Rightarrow$ to stabilize all ID's experiments : 48 BPm's(among the 120 ones) and 48 correctors positionned on the bellows:

- resolution 0.2 μm
- Bandwidth 0.1 to 100 Hz

Feed-forward compensation: Tables from magnetic measurements and experiments with beam applied on undulators correctors and possibly fast feedback correctors